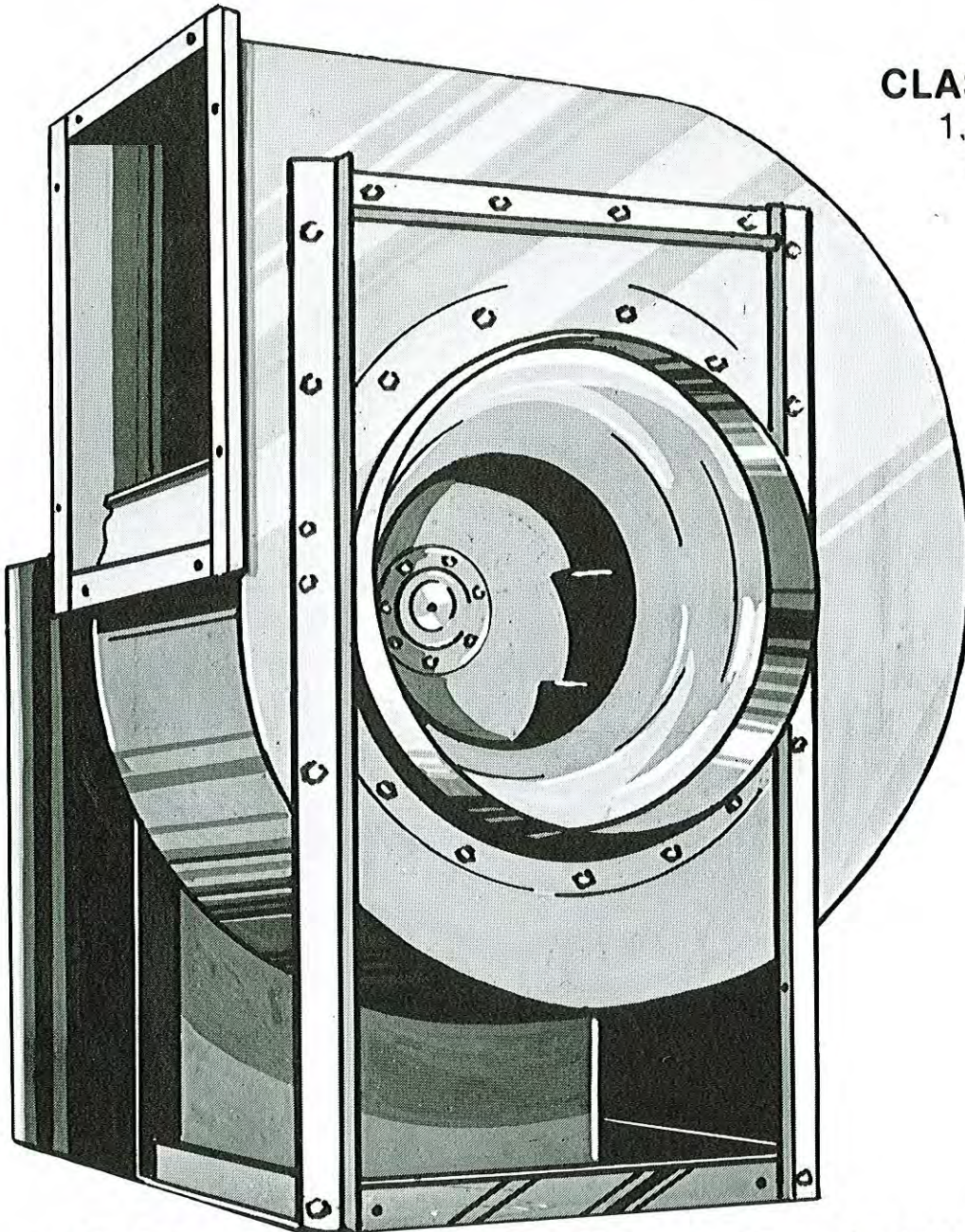


REVERSOMATIC

HEATING AND MANUFACTURING LIMITED

BACKWARD INCLINED BLADE CENTRIFUGAL FANS



CLASSES

1, 2 & 3

SIZES

30

33

36

40

44

49

54

60

**SINGLE WIDTH, SINGLE INTAKE
DOUBLE THICKNESS AIR FOIL BLADE**

REVERSOMATIC HEATING AND MANUFACTURING LTD. has realized the needs of the market for Backward Inclined fans in 30", 33", 36", 40", 44", 49", 54" & 60" wheel diameter ranges. We have developed these fans following strict engineering guidelines to provide rugged construction, high efficiency, low power requirements, along with stable performance and quiet operation.

AMCA CLASS LIMITS

The class operating limits have been defined at certain CFM and Static Pressure in accordance with AMCA standard 2408-69. Standard information in Class 1, 2 and 3 fans is provided in this catalogue.

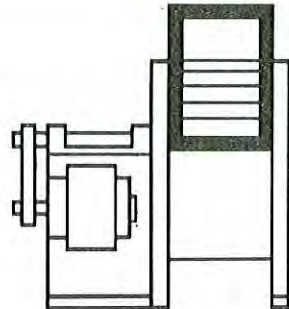
SOUND

Sound power levels are shown in a table on each performance curve for the area of maximum efficiency; sound levels outside this area can be approximately compensated by further addition of the indicated dB values for the lower efficiency areas, as defined by the radial lines on each performance curve.

FEATURES

- **ARRANGEMENT**

The Reversomatic Fan has been designed to comply with Arrangement No. 10, with blower wheel overhung, belt drive, two bearings and motor inside base. Alternate arrangements can be supplied on request.



Arrangement No. 10

- **SPUN INLET NOZZLE**

The nozzle is manufactured with heavy-gauge spun-steel to provide a smooth and quiet airflow into the blower wheel, assuring efficient performance at all operating levels.

- **HOUSING**

Fan scrolls are designed to provide optimum efficiency through all the CFM range. Housings are air tight sealed and braced with plate and angle to eliminate drumming and for increased rigidity and mechanical efficiency.

- **BLOWER WHEEL**

Wheels have stream-lined and double-thickness air foil blades. Blades are die-formed and jig-welded to the shroud and back plate to achieve maximum design efficiency.

Shrouds are spun to maintain minimum inlet clearance. All wheels have heavy duty cast iron hubs.

- **BEARING SUPPORTS**

Heavy duty bearing supports of structural steel prevent distortions due to belt pull and maintain accurate alignment of the moving parts.

- **SHAFT**

Shafts are machined or ground to the adequate allowances and tolerances that provide appropriate fits with bearings and hubs. Hubs are keyed to the shafts. Shafts are designed to tolerate at least 5% increase in speed over the maximum for each class limit.

- **BEARINGS**

Grease lubricated antifriction ball or roller bearings are standard. They are designed for an average life expectancy of at least 150,000 hours.

- **BALANCING**

Static and dynamic balancing, for both single plane and two plane components, assures quiet operation and maximum efficiency.

- **FLANGED OUTLETS**

Standard on all classes of construction.

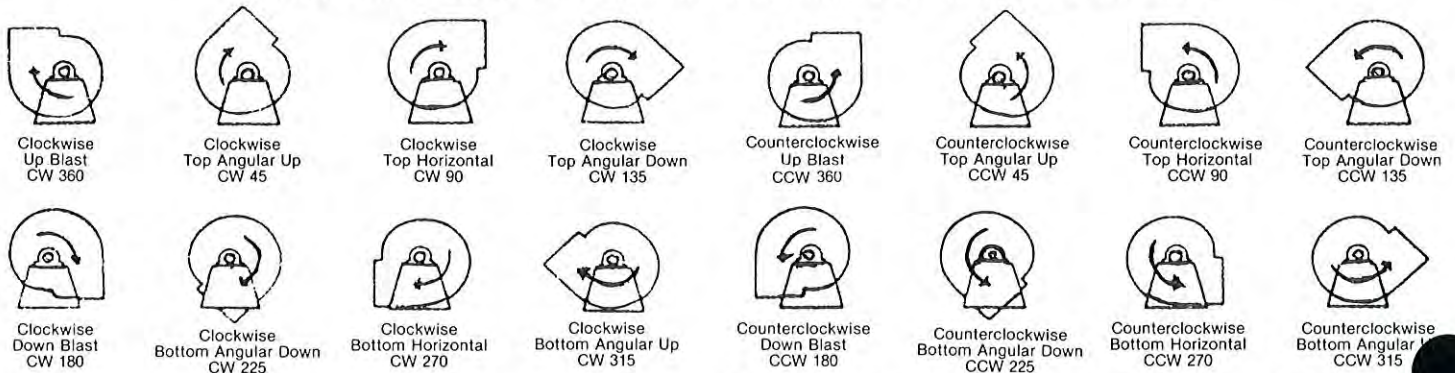
ACCESSORIES

- Access Doors on Fan Scrolls
- Plugged Drain in Lowest Part of Scroll
- Shaft Seals of Compressed Asbestos
- Inlet Bird Screen
- Intake and Exhaust Motorized Dampers
- Discharge Angles

APPLICATION

- To Pressurize Staircase & Elevator Shafts
- Air Make-Up
- Extraction of Fumes
- Kitchen Exhaust
- Dust Exhausting

DESIGNATIONS FOR ROTATION AND DISCHARGE OF CENTRIFUGAL FANS



Notes:

1. Direction of rotation is determined from drive side of fan.
2. On single inlet fans, drive side is always considered as the side opposite fan inlet.
3. On double inlet fans with drives on both sides, drive side is that with the higher powered drive under.

4. Direction of discharge is determined in accordance with diagrams. Angle of discharge is referred to the vertical axis of fan and designated in degrees from such standard reference axis. Angle of discharge may be any intermediate angle as required.
5. For fan inverted for ceiling suspension, or side wall mounting, direction of rotation and discharge is determined when fan is resting on floor.

START TIMES and WK²

Some low horsepower motors, when utilized to move large diameter wheels, might not be able to start the fan within a reasonable time. The next table indicates the different start times:

- 10 seconds or less: Satisfactory
- 11 to 15 seconds: Probably satisfactory
- 15 to 20 seconds: Check with starter and motor manufacturer
- Over 20 seconds: Not recommended

The start times of the fans shown in this catalogue can be approximately calculated using the following formula:

$$t = \frac{WK^2 \times N^2 (N/Nm)^2}{1.62 + HPm}$$

t = Start time in seconds

WK² = Blower wheel moment of inertia lb x ft²

N = Fan speed in 1000's of rpm

Nm = Motor speed in 1000's of rpm

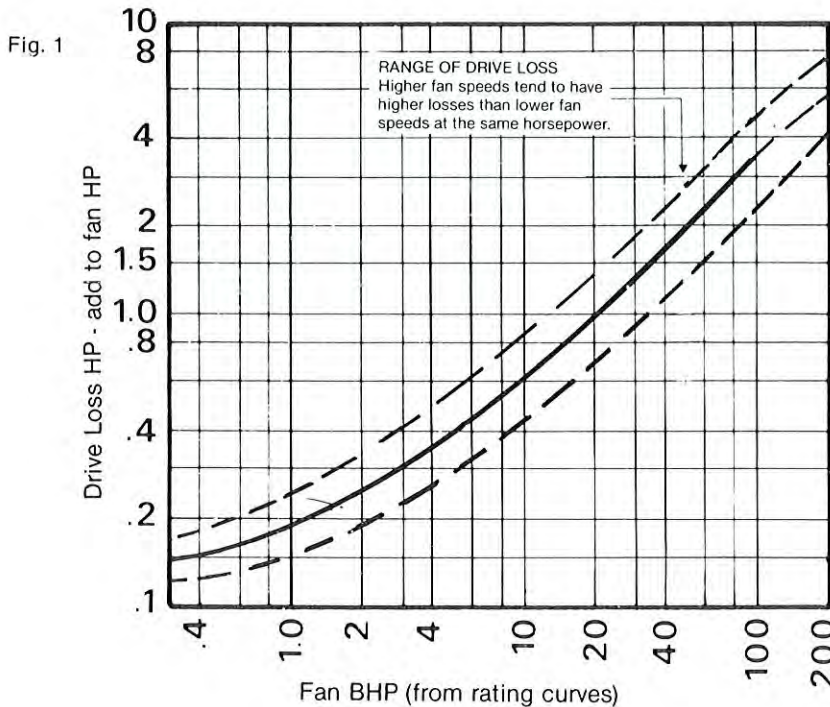
HPm = Motor Horsepower

WK² Values, lb. ft²

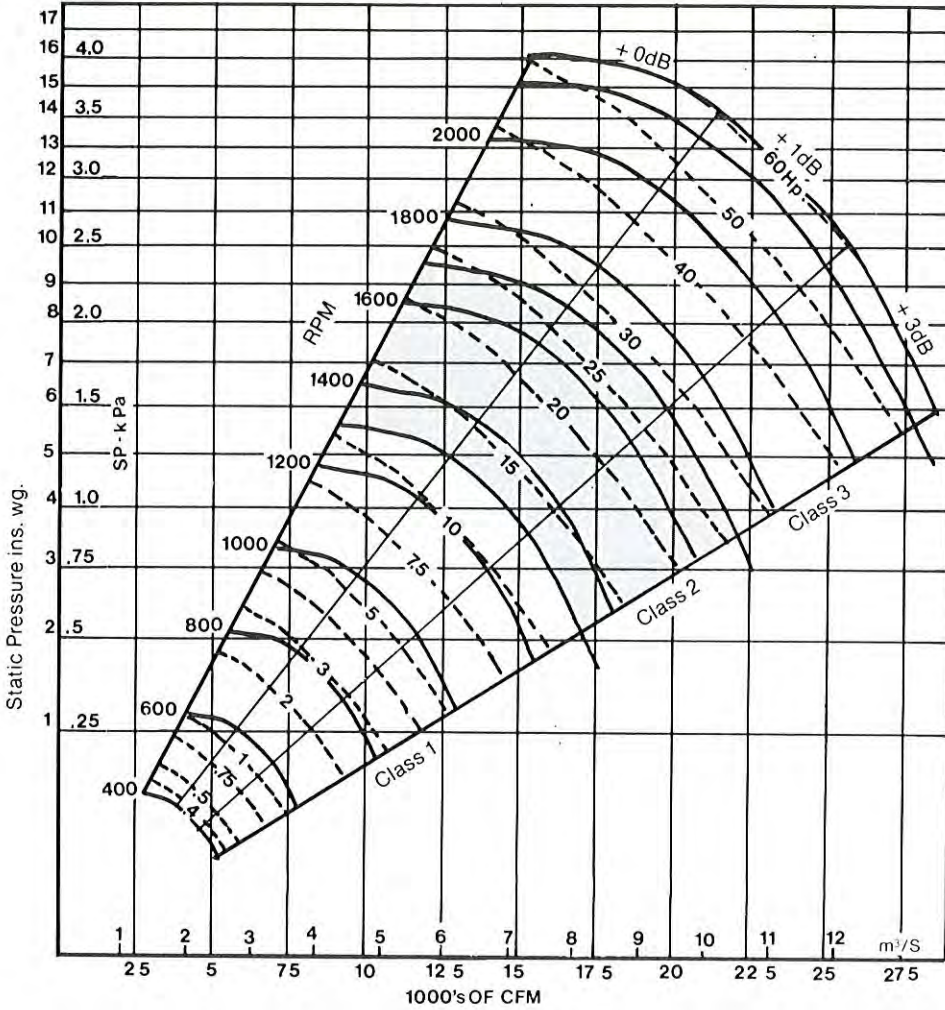
MODEL	CLASS 1	CLASS 2	CLASS 3
30	97	97	118
33	152	152	176
36	218	218	255
40	365	365	431
44	732	732	870
49	1116	1116	1268
54	1733	1733	1809
60	2845	2845	3536

DRIVE LOSSES

Belt driven fans require a horsepower allowance for drive losses. This allowance must be added to the horsepower obtained from the rating curves. The curve shown in figure 1 is based on test and experience and indicates the horsepower to be added to the fan for compensation of drive losses.



Model 30



BHP shown does not include drive losses. For drive losses see Fig. 1

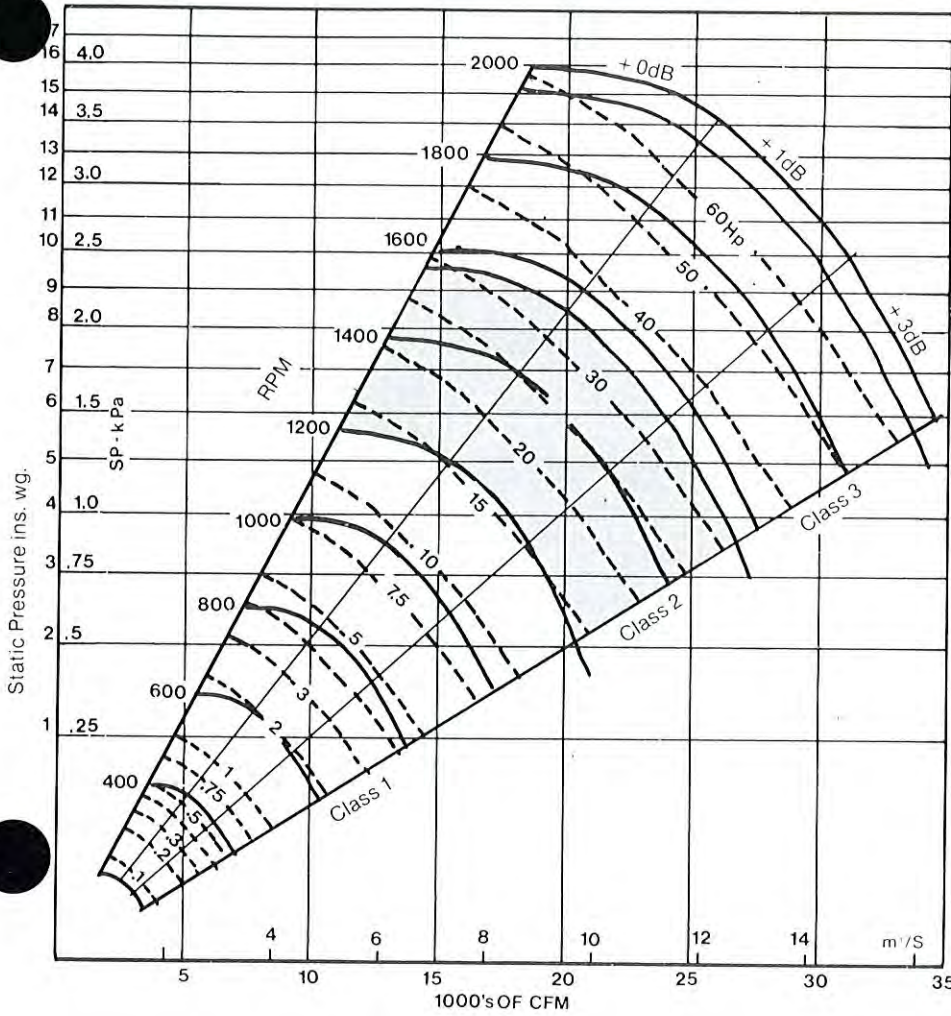
Performance shown above is for fans with outlet ducts and with or without inlet ducts, based on test results obtained from Arr. 1 fans.

WHEEL DIA. = 30"
 O.A. = 5.17 ft²
 MAX. BHP = $5.66 \left(\frac{\text{RPM}}{1000}\right)^3$

RPM	Octave Bands Mid Frequency -- Hz							
	63	125	250	500	1000	2000	4000	8000
400	70	63	59	59	59	57	50	44
600	83	75	69	67	69	68	62	56
800	91	85	78	74	74	75	71	65
1000	97	93	84	79	78	80	78	72
1200	101	98	90	84	82	84	83	78
1400	104	102	96	89	86	87	86	82
1600	106	106	100	93	89	89	89	87
1800	108	109	104	96	92	91	92	90
2000	110	112	108	100	94	93	95	93
2200	112	115	111	103	97	96	97	96

Volume CFM	O.Vel FPM	1" SP		2" SP		4" SP		6" SP		8" SP		10" SP		12" SP		14" SP		16" SP	
		RPM	BHP	RPM	BHP	RPM	BHP	RPM	BHP	RPM	BHP	RPM	BHP	RPM	BHP	RPM	BHP	RPM	BHP
3500	677																		
4000	774	527	0.80																
4500	870	546	0.91																
5000	967	567	1.04																
5500	1064	589	1.19	740	2.18														
6000	1161	612	1.35	758	2.40														
6500	1257	635	1.54	777	2.64														
7000	1354	660	1.75	799	2.89														
7500	1451	688	1.98	821	3.18	1037	5.93												
8000	1518	718	2.24	843	3.49	1053	6.35												
8500	1644	748	2.52	866	3.63	1071	6.79												
9000	1741	780	2.83	888	4.20	1090	7.26	1264	10.65										
9500	1838	812	3.17	912	4.60	1111	7.76	1279	11.26										
10000	1934	845	3.54	938	5.04	1133	8.28	1295	11.90										
11000	2128	912	4.37	996	5.99	1177	9.45	1333	13.27	1476	17.36								
12000	2321			1057	7.09	1222	10.77	1374	14.78	1510	19.08	1639	23.60						
13000	2515			1120	8.35	1267	12.26	1418	16.47	1549	20.98	1672	25.72	1790	30.64				
14000	2708			1185	9.77	1317	13.95	1462	18.32	1591	23.02	1709	27.97	1821	33.11	1930	38.42		
15000	2902			1251	11.38	1373	15.80	1509	20.41	1635	30.43	1749	30.43	1857	35.79	1961	41.29	2062	46.97
16000	3095			1318	13.17	1433	17.84	1554	22.66	1679	33.06	1793	33.06	1896	38.62	1996	44.34		
17000	3288					1494	20.08	1604	25.18	1725	35.92	1836	35.92	1938	41.66	2035	47.64		
18000	3482					1557	22.55	1658	27.90	1770	39.02	1880	39.02	1982	44.94	2076	51.13		
19000	3675					1621	25.24	1717	30.83	1818	42.40	1926	42.40	2026	48.50	2119	54.80		
20000	3869					1687	28.19	1777	34.02	1869	46.02	1971	52.29	2070	52.29				
22000	4256					1820	34.87	1902	41.16	1983	54.16	2068	60.77	2161	60.77				
24000	4643							2031	49.45	2105	63.42	2179	70.51						

Model 33



BHP shown does not include drive losses. For drive losses see Fig. 1

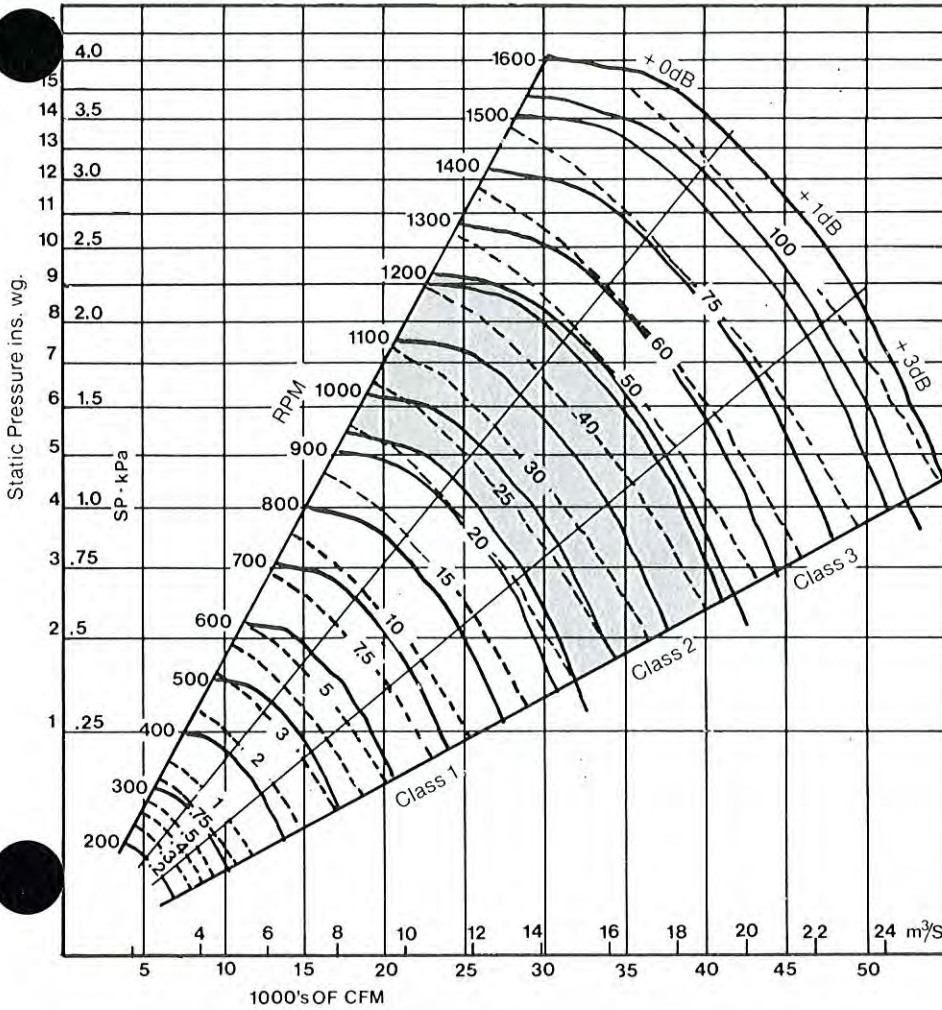
Performance shown above is for fans with outlet ducts and with or without inlet ducts, based on test results obtained from Arr. 1 fans.

WHEEL DIA. = 33"
 O.A. = 6.26 ft²
 MAX. BHP = 9.12 $\left(\frac{RPM}{1000}\right)^3$

RPM	Octave Bands Mid Frequency — Hz							
	63	125	250	500	1000	2000	4000	8000
200	51	47	47	47	44	39	31	25
400	73	66	62	62	62	60	53	47
600	86	78	72	70	72	71	65	59
800	94	88	81	77	77	78	74	68
1000	100	96	87	82	81	83	81	75
1200	103	101	93	87	85	87	86	81
1400	106	105	99	92	89	90	89	85
1600	108	109	103	96	92	92	92	89
1800	111	112	107	99	95	94	95	93
2000	112	115	111	102	97	96	98	96

Volume CFM	O.Vel FPM	1" SP		2" SP		4" SP		6" SP		8" SP		10" SP		12" SP		14" SP		16" SP	
		RPM	BHP	RPM	BHP	RPM	BHP	RPM	BHP	RPM	BHP	RPM	BHP	RPM	BHP	RPM	BHP	RPM	BHP
5000	799	479	1.00																
5500	879	492	1.11																
6000	959	506	1.23																
6500	1038	522	1.37	665	2.60														
7000	1118	538	1.53	676	2.79														
7500	1198	556	1.71	688	3.01														
8000	1278	575	1.91	701	3.23														
8500	1358	595	2.12	715	3.49														
9000	1438	616	2.36	731	3.77	936	7.19												
9500	1518	638	2.61	747	4.06	946	7.56												
10000	1598	660	2.89	764	4.39	957	7.97												
11000	1757	707	3.53	801	5.14	982	8.84	1145	13.16										
12000	1917	756	4.27	840	5.97	1010	9.82	1165	14.30										
13000	2077	805	5.12	882	6.93	1041	10.94	1189	15.58	1327	20.67								
14000	2237	856	6.09	927	8.01	1075	12.21	1215	16.95	1347	22.21								
15000	2397			973	9.22	1110	13.63	1245	18.50	1372	23.93	1491	29.70						
16000	2556			1021	10.58	1148	15.20	1276	20.19	1397	25.73	1513	31.74	1623	38.04				
17000	2716			1069	12.10	1187	16.91	1310	22.08	1426	27.74	1537	33.86	1644	40.33	1746	47.09		
18000	2876			1119	13.77	1229	18.80	1345	24.15	1457	29.93	1564	36.16	1667	42.80	1766	49.75	1862	56.96
19000	3036			1169	15.62	1273	20.84	1381	26.45	1489	32.31	1592	38.67	1692	45.44	1788	52.59		
20000	3195			1220	17.53	1318	23.06	1419	28.86	1523	34.92	1623	41.38	1719	48.28	1812	55.56		
22000	3515					1412	28.11	1502	34.32	1595	40.83	1688	47.47	1779	54.61	1866	62.10		
24000	3834					1508	34.00	1590	40.56	1673	47.46	1759	54.57	1844	61.87				
26000	4154					1608	40.79	1682	47.75	1757	55.05	1835	62.58	1914	70.27				
28000	4473					1709	48.58	1777	55.95	1846	63.59	1917	71.62						
30000	4793							1875	65.26	1938	73.23	2003	81.57						

Model 40



BHP shown does not include drive losses. For drive losses see Fig. 1

Performance shown above is for fans with outlet ducts and with or without inlet ducts, based on test results obtained from Arr. 1 fans.

WHEEL DIA. = 40 1/4"

O.A. = 9.31 ft²

MAX. BHP = 26.96 $\left(\frac{\text{RPM}}{1000}\right)^3$

RPM	Octave Bands Mid Frequency — Hz							
	63	125	250	500	1000	2000	4000	8000
200	55	53	48	46	45	39	33	28
300	69	63	59	56	54	51	46	40
400	81	70	68	63	61	60	54	49
500	88	77	73	69	66	66	61	55
600	93	84	78	74	71	70	66	61
700	97	91	82	79	74	73	71	65
800	101	96	85	83	78	76	75	69
900	104	100	88	86	81	78	78	73
1000	106	103	92	89	84	81	81	76
1100	107	106	96	91	87	83	83	79
1200	109	108	99	93	89	86	85	82
1300	110	110	103	95	92	88	86	84
1400	111	112	106	97	94	89	88	86
1500	113	114	108	98	96	91	89	88
1600	114	116	111	100	98	93	91	90

Volume CFM	O.Vel FPM	1" SP		2" SP		4" SP		6" SP		8" SP		10" SP		12" SP		14" SP		16" SP	
		RPM	BHP	RPM	BHP	RPM	BHP	RPM	BHP	RPM	BHP	RPM	BHP	RPM	BHP	RPM	BHP	RPM	BHP
7000	752	388	1.4																
8000	859	401	1.6																
9000	967	416	1.9																
10000	1074	434	2.1																
11000	1181	453	2.5	610	6.0														
12000	1289	473	2.9	629	6.6														
13000	1396	496	3.3	649	7.4														
14000	1504	520	3.8	670	8.2														
15000	1611	545	4.4	692	9.0	768	12.0												
16000	1718	570	5.0	715	10.0	800	12.8												
17000	1826	597	5.7	739	11.0	815	13.8	945	20.3										
18000	1933	624	6.5	764	12.1	831	14.8	957	21.5										
19000	2041	651	7.3	790	13.3	848	15.9	970	22.7										
20000	2148	679	8.2	842	16.0	866	17.1	984	24.1										
21000	2255	707	9.2	896	19.2	884	18.4	999	25.5	1107	33.3								
22000	2363	735	10.3	951	22.7	904	19.8	1015	27.0	1120	35.0								
24000	2577	792	12.8	1007	26.8	845	22.9	1050	30.4	1149	38.7	1243	47.6						
26000	2792	849	15.7	1064	31.3	990	26.5	1087	34.3	1181	42.8	1271	52.0	1357	61.8				
28000	3007			1121	36.4	1037	30.4	1127	38.7	1216	47.4	1301	56.8	1383	66.9	1483	77.4		
30000	3222			1178	42.0	1087	34.9	1169	43.6	1253	52.6	1335	62.3	1413	72.6	1489	83.4		
32000	3437			1236	48.2	1138	39.9	1214	49.0	1293	58.4	1371	68.3	1446	78.8	1518	89.9	1589	101.4
34000	3651			1294	55.1	1191	45.4	1261	54.9	1334	64.8	1408	74.9	1480	85.6	1551	97.0		
36000	3866					1245	51.5	1311	61.3	1378	71.7	1448	82.3	1517	93.2	1584	104.7		
38000	4081					1299	58.2	1361	68.5	1425	79.2	1490	90.3	1556	101.5	1620	113.2		
40000	4296					1355	65.7	1413	76.3	1473	87.4	1534	98.8	1598	110.7				
44000	4725					1467	82.5	1520	94.0	1573	105.8	1627	118.1						

DRIVE SELECTION

- Up to 3 H.P. One Belt
Light Duty Drive
- 5 to 14 H.P. Double Belt
Medium Duty Drive
- 15 to 50 H.P. Triple Belt
Heavy Duty Drive

THE BELTS ARE "V" SHAPED OF "A" AND "B" CROSS SECTION AREAS' DESIGNED TO PROVIDE SUPER-RATED PERFORMANCE WITH SMALL SHEAVES TORQUE-FLEX BELTS ARE PROVIDED TO OFFSET BINDING STRESSES.

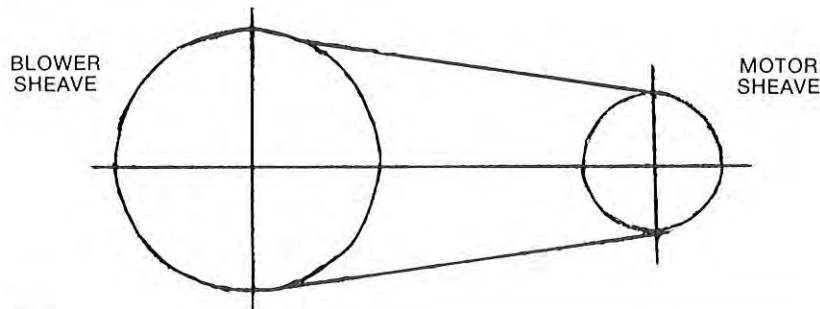
MINIMUM RECOMMENDED SHEAVE DIAMETERS IN INCHES FOR ELECTRIC MOTORS

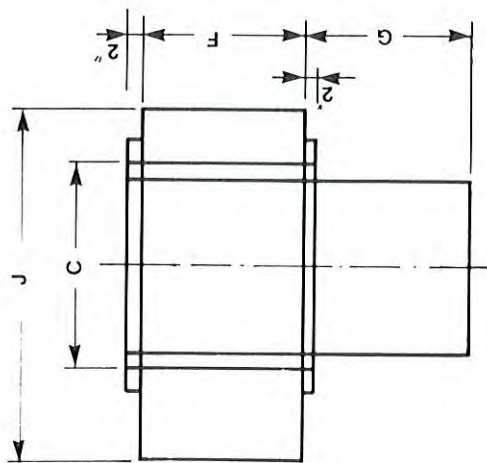
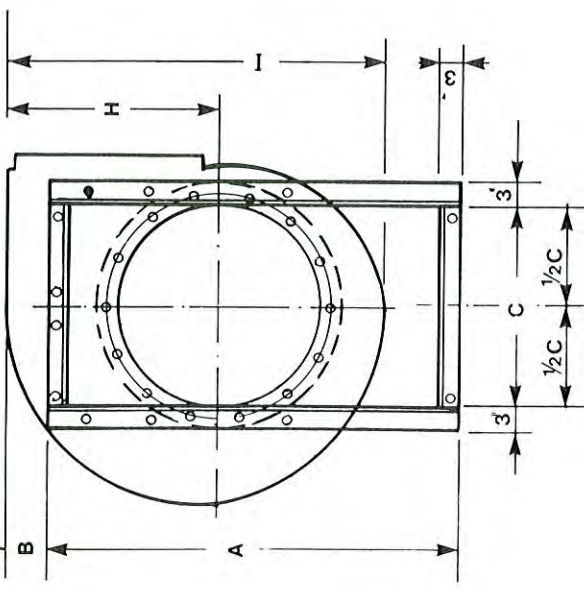
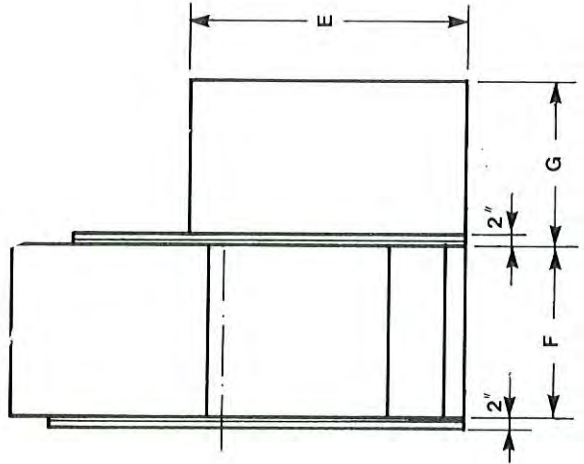
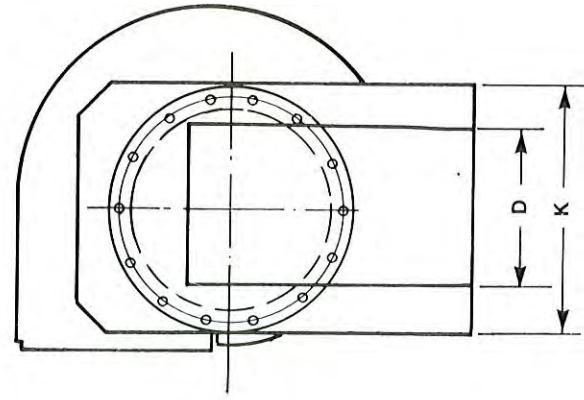
Motor Horse-Power	MOTOR RPM					
	575	695	870	1160	1750	3450
1/2	2-1/4	2-1/4
3/4	2-1/2	2-1/2
1	3	2-1/2	2-1/2	2-1/2	2-1/4	...
1-1/2	3	3	3	2-1/2	2-1/2	2-1/4
2	3-3/4	3	3	3	2-1/2	2-1/2
3	4-1/2	3-3/4	3	3	2-1/2	2-1/2
5	4-1/2	4-1/2	3-3/4	3-3/4	3	2-1/2
7-1/2	5-1/4	4-1/2	4-1/2	4-1/2	3	3
10	6	5-1/4	4-1/2	4-1/2	3-3/4	3
15	6-3/4	6	5-1/4	5-1/4	4-1/2	3-3/4
20	8-1/4	6-3/4	6	6	4-1/2	4-1/2
25	9	8-1/4	6-3/4	6-3/4	4-1/2	4-1/2
30	10	9	6-3/4	6-3/4	5-1/2	...
40	10	10	8-1/4	8-1/4	6	...
50	11	10	9	9	6-3/4	...
60	12	11	10	10	7-1/2	...
75	14	13	10	13	9	...
100	18	15	13	13	10	...
125	20	18	15	13	11	...
150	22	20	18
200	22	22	22
250	22	22
300	27	27

$$D_B = \frac{D_M \times \text{RPM}_M}{\text{RPM}_B}$$

WHERE
 D_B — BLOWER SHEAVE DIAMETER
 D_M — MOTOR SHEAVE DIAMETER
 RPM_M — MOTOR SPEED
 RPM_B — BLOWER SPEED

NOTE: Data in the clear section is from National Electrical Manufacturers Association Standard MGI-14-45. Data in the shaded section is a composite of Electrical Motor Manufacturers. They are generally conservative, and specific motors and bearings may permit the use of a smaller motor sheave. Consult the motor manufacturer.





DIMENSION	30	33	36	40	44	49	54	60
A	63	68	75	79	87	99	110	121
B	6	7	8	9	10	11	12	13
C	30	33	37	40-3/4	45-1/4	50	55	61
D	24-1/2	24-1/2	24-1/2	24-1/2	24-1/2	24-1/2	24-1/2	24-1/2
E	45	48	52	57	63	68	74	80
F	23	25-1/2	28	31-1/4	34-1/2	38	42	46
G	41	41	41	41	41	41	41	41
H	32-1/2	36	39-1/2	43-1/2	48-1/2	53	59	65
I	57-1/2	63	69-3/4	78	86	95	105	116
J	53	58	64	70	77	84	93	103
K	36-1/2	39-1/2	43	46-3/4	51-1/4	56	61	67

DIMENSIONS IN INCHES

JOB	FAN		FAN
	DETAIL		TYPE BI--
CONTRACTOR	MOTOR		DATE
	ENGINEER/ARCHITECT		SUPERSEDES
DRAWING NO		DRAWING NO	
REVERSONAL Heating & Mfg. Ltd.		REVERSONAL Heating & Mfg. Ltd.	
Toronto, Canada		Toronto, Canada	



REVERSOMATIC
MANUFACTURING LIMITED

INSTALLATION & MAINTENANCE GUIDE

General Instructions

REGULAR MAINTENANCE / LUBRICATION OF THIS UNIT IS REQUIRED TO MAINTAIN THE MANUFACTURER'S WARRANTY.

FOR MOTORS LARGER THAN 7 ½ HP, AN ELECTRONIC "SOFT START" CONTROL IS RECOMMENDED FOR LONGER BELT LIFE.

C A U T I O N

IT IS STRONGLY RECOMMENDED THAT BEFORE STARTING UP THE FAN THE FOLLOWING INSPECTIONS ARE PERFORMED.

- 1. FAN BELT (IF USED) TIGHTNESS AND ALIGNMENT.**
- 2. FAN BLADE CENTERING AND ROTATION.**
- 3. FAN BLADE MOUNTING BOLT TIGHTNESS.**
- 4. MOTOR MOUNTING PLATE BOLT TIGHTNESS.**
- 5. BEARING LUBRICATION.**

WARNINGS AND SAFETY INSTRUCTIONS

1. Do not operate the fan excess of maximum limit.
2. Do not permit any object to enter the fan inlets or outlets; provide a screen covering.
3. Do not operate the fan without adequate guards over rotating parts; provide drive belt, coupling and shaft guards.
4. Provide a disconnect switch with a padlock to prevent fan switch use during maintenance.
5. Locate a disconnect switch at the fan for use of personnel working on the fan.
6. Provide vibration limiting switches to detect sudden changes in the operation of the fan, especially when operating a fan under high temperatures or in an extremely corrosive atmosphere such as fly ash.
7. Lubricate and service bearings regularly. see lubrication schedule.

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INSTALLATION & MAINTENANCE GUIDE

Performance Problems:

CFM too low - These are some common sources of this problem:

Fan - forward curved impeller installed backwards, fan running backwards, cutoff missing or improperly installed, impeller not centered with inlet collars, fan speed too slow.

System - more resistant to flow than expected, dampers or registers closed, leaks in supply ducts, insulating duct liner loose, clogged filters or coils.

Fan Inlets - leaks around fan inlets, elbows near the inlet, cabinet walls too close. Inlet obstructions cause more restrictive systems but do not cause increased negative pressure readings near the fan inlets. Fan speed may be increased to counteract the effect of restricted fan inlet, but check the maximum RPM for the wheel construction before increasing the speed.

Fan Outlet - most centrifugal fans are used in ducted systems and have been tested with a length of straight duct at the fan outlet, If there is no straight duct at the fan outlet, decreased fan performance will result. If it is not practical to install a straight section of duct at the fan outlet, the fan speed may be increased to overcome this pressure loss. Other sources affecting fan outlet may be sharp elbow nearby, improperly designed turning vanes or other obstructions near the outlet.

Noise - may be caused by:

- Impeller hitting the inlet of the fan or cutoff plate, loose impeller.
- Drives can cause noise if sheaves are not tight on the shaft, belts are too loose or too tight, wrong belt cross section, or mis-matched belts, also worn belts, oily belts or mis-aligned sheaves.
- If couplings are used they may be source of noise by being unbalanced, misaligned, loose or dry of lubricant.
- Bad bearings are a common source of noise when defective, dry of lubricant, loose on the bearing support, loose on the shaft, seals mis-aligned, dirty lubricant, fretting corrosion between inner race and shaft, etc. See separate section on bearing care.
- There can be an electrical source of noise such as AC hum in motor or relay, starting relay chatter, noisy motor bearings, single phasing a 3 phase motor, etc.
- A bent or undersized shaft may be a noise source. **IF MORE THAN TWO BEARINGS ARE ON THE SAME SHAFT, THEY MUST BE CAREFULLY ALIGNED.**
- There may be other noise sources such as obstruction in high velocity air stream causing rattle or pure tone whistle, fan operating at undesirable design point, causing pulsation, cracks or holes in duct work, or whistles in fan housing.

LUBRICATION

RELUBRICATION SCHEDULE (MONTHS)* Ball Bearing Pillow Blocks

SHAFT DIAMETER	Operating Speed (RPM)									
	500	1000	1500	2000	2500	3000	3500	4000	4500	5000
5/8" thru 1"	6	6	6	6	4	4	4	4	2	2
1 1/8" thru 1 1/2"	6	6	6	4	4	4	2	2	2	1
1 5/8" thru 1 15/16"	6	6	6	4	4	2	2	1	1	-
2" thru 2 1/2"	6	6	4	4	2	1	1	-	-	-
2 11/16" thru 3 3/16"	6	4	2	2	1	1	1/2	-	-	-

*Suggested initial greasing interval - relubricate while running, if safety permits, until some purging occurs at seals. Adjust lubrication frequency depending on condition of purged grease. Hours of operation, temperature, and surrounding conditions will affect the relubrication frequency required.

- Lubricate with a high quality NLGI No. 2 or No.3 multi purpose ball bearing grease having rust inhibitors and anti-oxidant additives.

Some grease having these properties are:

- Shell - *Alvania EP No.2*
- Mobil - *Mobilux Ep2*
- Texaco - *Molytex No.2*
- Gulf - *Gulfcrown No. 2*
- American - *Amolith No. 2*

- Lubricate bearings prior to extended shutdown or storage and rotate shaft monthly to avoid corrosion.

Spherical Roller Bearings - Solid Pillow Blocks

SHAFT DIAMETER	Operating Speed (RPM)									
	500	1000	1500	2000	2200	2700	3000	3500	4000	4500
3/4" thru 1"	6	6	6	4	4	4	2	2	1	1
1 1/8" thru 1 1/4"	6	6	4	4	2	2	1	1	1	1
1 7/16" thru 1 1/2"	6	4	4	2	2	1	1	1	1	1/2
1 5/8" thru 1 3/4"	6	4	2	2	1	1	1	1	1/2	-
1" 15/16" thru 2"	6	4	2	1	1	1	1	1/2	-	-
2 3/16" thru 2 1/4"	6	4	2	1	1	1	1/2	-	-	-
2 7/16" thru 2 1/2"	4	2	1	1	1	1/2	-	-	-	-
2 11/16" thru 3"	4	2	1	1	1/2	-	-	-	-	-
3 3/16" thru 3 1/2"	4	2	1	1/2	-	-	-	-	-	-

- Lubricate with a multi-purpose roller bearing NLGI grade 2 grease having rust inhibitors, anti-oxidant additives, and a minimum oil viscosity of 500 SSU at 100°F. Some additives.

Some grease having these properties are:

- American - *Rykon No.2*
- Mobil - *Mobilgrease 28*
- Texaco - *Molytex Ep2 grease*

- Lubricate bearings prior to extended shutdown or storage and rotate shaft monthly to avoid corrosion.

RECOMMENDED 'SKF' GREASES FOR 'SKF' BEARINGS

Fixed Pillow Block - LGMT2 Fans running below 80°C (176°F)

Split Pillow Block - LGMT3 Fans running below 80°C (176°F)

Fans Running Above 80°C thru 150°C - LGHT3

TROUBLESHOOTING LIST

- IMPELLER** a. Loose on shaft
b. Unbalance
- DRIVE** a. Sheave not tight on shaft (motor or fan)
b. Belts hitting belt tube or belt guard
c. Belts too loose. Adjust for belt stretching after 48 hours of operation.
d. Belts too tight
e. Belts wrong cross-section
f. Belts not "Matched" in length on multi-belt drive
g. Variable pitch sheaves not adjusted so each groove has same pitch diameter (multi-belt drive)
h. Misaligned sheaves
i. Belts worn
j. Motor, Motor base or fan not securely anchored
k. Belts oily or dirty
l. Improper drive selection
m. loose key
n. Excessive start-stop cycles
- COUPLING** a. Coupling unbalanced, misaligned, loose or may need lubricant
b. Loose key
- BEARING** a. Defective bearing
b. Needs lubrication
c. Loose on bearing support
d. Loose on shaft
e. Seals misaligned
f. Foreign material inside bearing
g. Worn bearing
h. Fretting corrosion between inner race and shaft
i. Bearing not sitting on flat surface
j. Excessive belt tension
- SHAFT** a. Bent
b. Undersized
- MOTOR** a. Noisy motor bearings
b. Single phasing a three phase motor
c. Low voltage
- LOOSE FASTENERS** a. Impeller set screws
b. Bearing set screws
c. Drive component set screws
d. Fan mounting bolts
e. Bearing bolts
f. Motor bolts



WARRANTY

REVERSOMATIC HEATING AND MANUFACTURING LIMITED WARRANTS IT WILL PROVIDE A REPLACEMENT PART OF ITS FANS FOUND TO BE DEFFECTIVE IN MATERIAL OR WORKMANSHIP FOR A PERIOD OF ONE YEAR FROM DATE OF PURCHASE FOR FIRST USER, F.O.B. OUR PLANT.

NOTE: THIS WARRANTY DOES NOT APPLY TO LABOUR COSTS INVOLVED IN REPLACEMENT OR REINSTALLATION, DIAGNOSTIC SERVICE, CLEANING AND ADJUSTMENT, OR TRANSPORTATION. FURTHERMORE WARRANTY IS ONLY APPLICABLE WHEN REVERSOMATIC REPLACEMENT PARTS AND ACCESSORIES ARE USED.

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